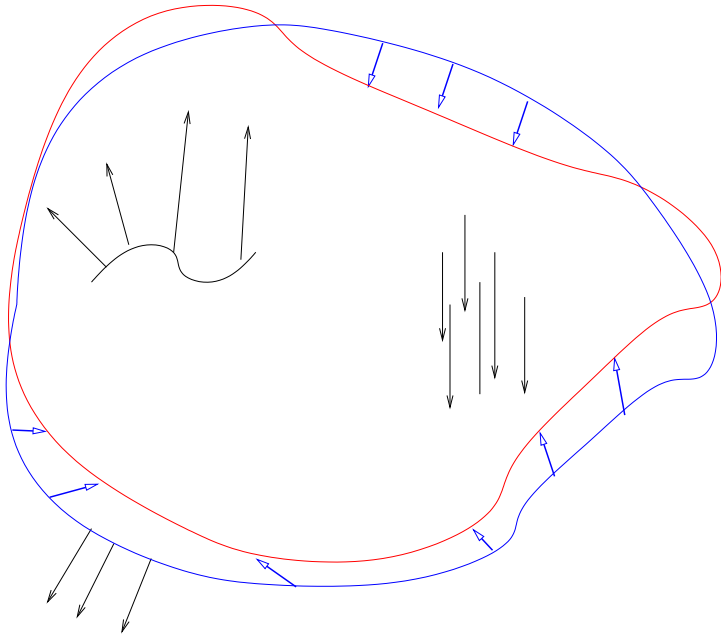


3D-2D Analysis for the optimal elastic compliance problem

G. Bouchitté, I. Fragala and P. Seppecher

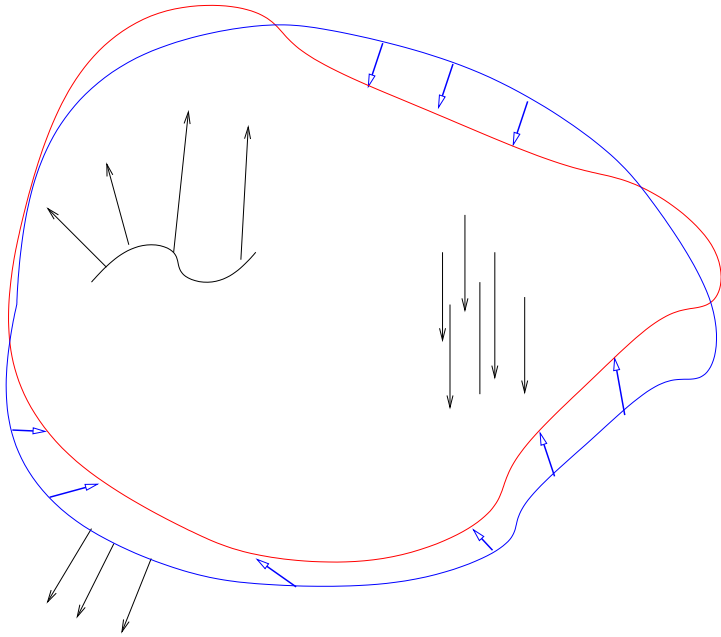
Toulon (FRANCE), Milano (Italie)

The framework of linear elasticity



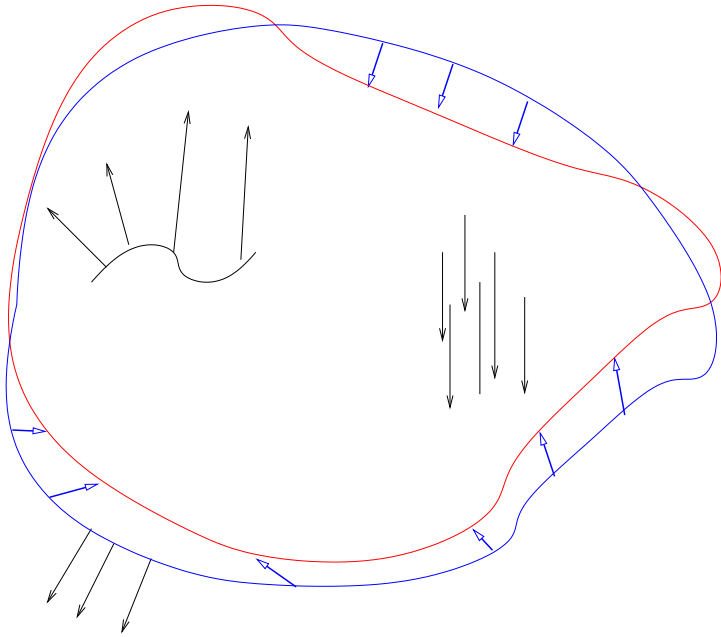
- A displacement field u on Ω

The framework of linear elasticity



- A displacement field u on Ω
- The (linearized) strain tensor field $e(u) = (\nabla u + \nabla u^T)/2$

The framework of linear elasticity



- A displacement field u on Ω
- The (linearized) strain tensor field $e(u) = (\nabla u + \nabla u^T)/2$
- A quadratic coercive energy density $j(e(u))$
(e.g. $j(e(u)) = \lambda(\text{tr}(e(u)))^2 + \mu/2 \|e(u)\|^2$)

The framework of linear elasticity

- A force distribution F (volume or surface density, inside Ω or on its boundary)

The framework of linear elasticity

- A force distribution F (volume or surface density, inside Ω or on its boundary)
- $u \in H^1(\Omega)$, $F \in H^{-1}(\Omega)$.

The framework of linear elasticity

- A force distribution F (volume or surface density, inside Ω or on its boundary)
- $u \in H^1(\Omega)$, $F \in H^{-1}(\Omega)$.

The equilibrium problem

$$\operatorname{div} \frac{\partial j}{\partial e}(e(u)) + F = 0 \text{ on } \Omega$$

The framework of linear elasticity

- A force distribution F (volume or surface density, inside Ω or on its boundary)
- $u \in H^1(\Omega)$, $F \in H^{-1}(\Omega)$.

The equilibrium problem

$$\operatorname{div} \frac{\partial j}{\partial e}(e(u)) + F = 0 \text{ on } \Omega$$

$$-C = \inf_u \left\{ \int_{\Omega} j(e(u)) - \langle F, u \rangle \right\}$$

$$C = \sup_u \left\{ \langle F, u \rangle - \int_{\Omega} j(e(u)) \right\}$$

The framework of linear elasticity

C is the “compliance” = the stored energy at equilibrium

- C is positive, depends on Ω , j and F .

The framework of linear elasticity

C is the “compliance” = the stored energy at equilibrium

- C is positive, depends on Ω , j and F .
- C is finite if F is balanced ($\langle F, u \rangle = 0$ whenever u is rigid) ($\int F = 0, \int x \wedge F = 0$)

The framework of linear elasticity

C is the “compliance” = the stored energy at equilibrium

- C is positive, depends on Ω , j and F .
- C is finite if F is balanced ($\langle F, u \rangle = 0$ whenever u is rigid) ($\int F = 0$, $\int x \wedge F = 0$)
- A solution \bar{u} exists (unique up to rigid motions)

The framework of linear elasticity

C is the “compliance” = the stored energy at equilibrium

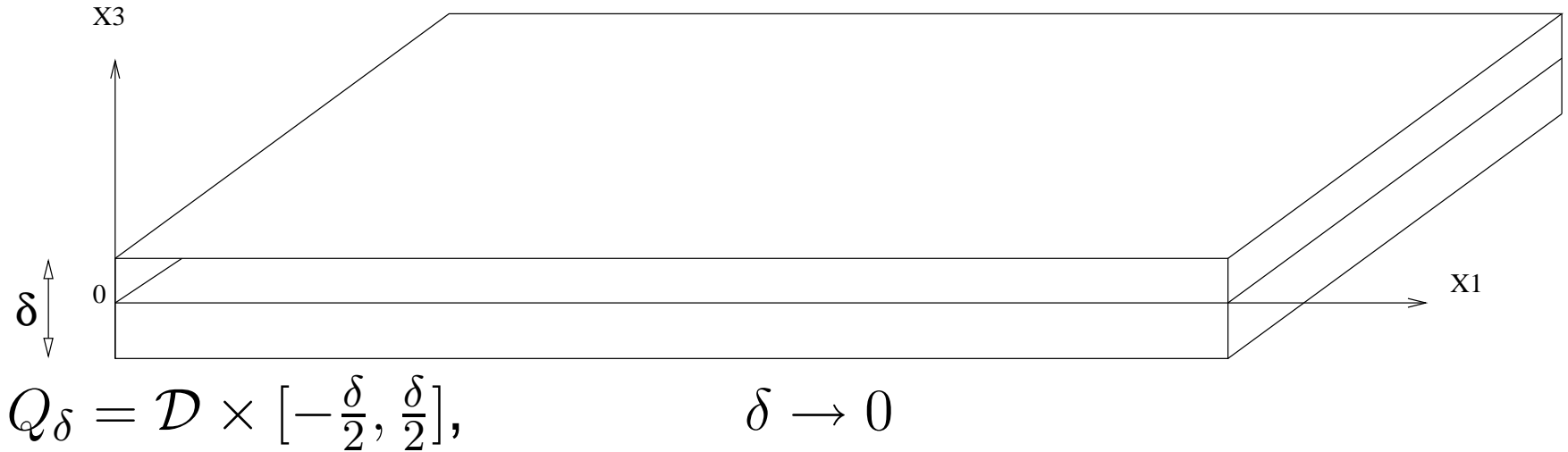
- C is positive, depends on Ω , j and F .
- C is finite if F is balanced ($\langle F, u \rangle = 0$ whenever u is rigid) ($\int F = 0$, $\int x \wedge F = 0$)
- A solution \bar{u} exists (unique up to rigid motions)

Dual formulation

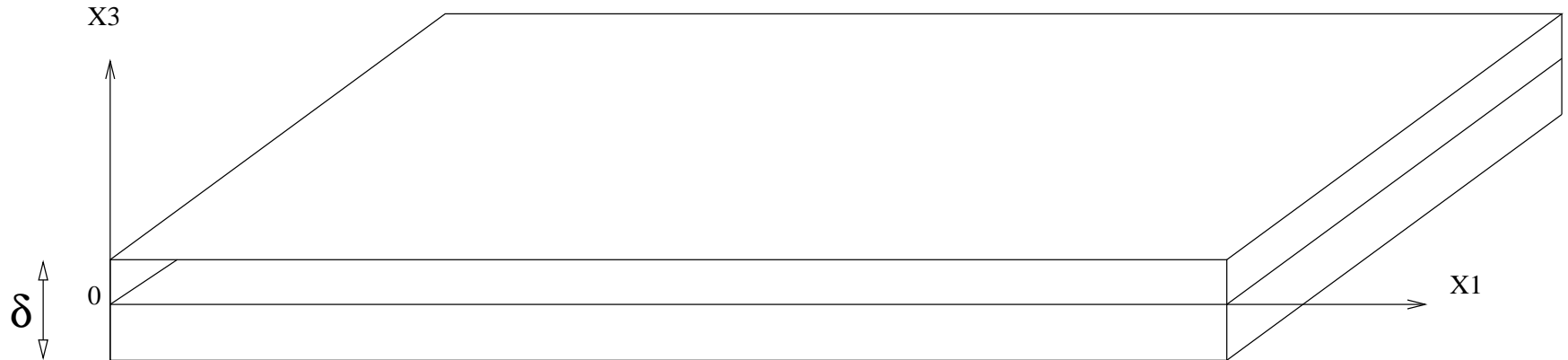
$$C = \inf_{\Sigma} \left\{ \int_{\Omega} j^*(\Sigma); \operatorname{div}(\Sigma) + F = 0 \right\}$$

$$j^*(A) := \sup_B \{ A \cdot B - j(B) \}$$

3D-2D asymptotic analysis



3D-2D asymptotic analysis

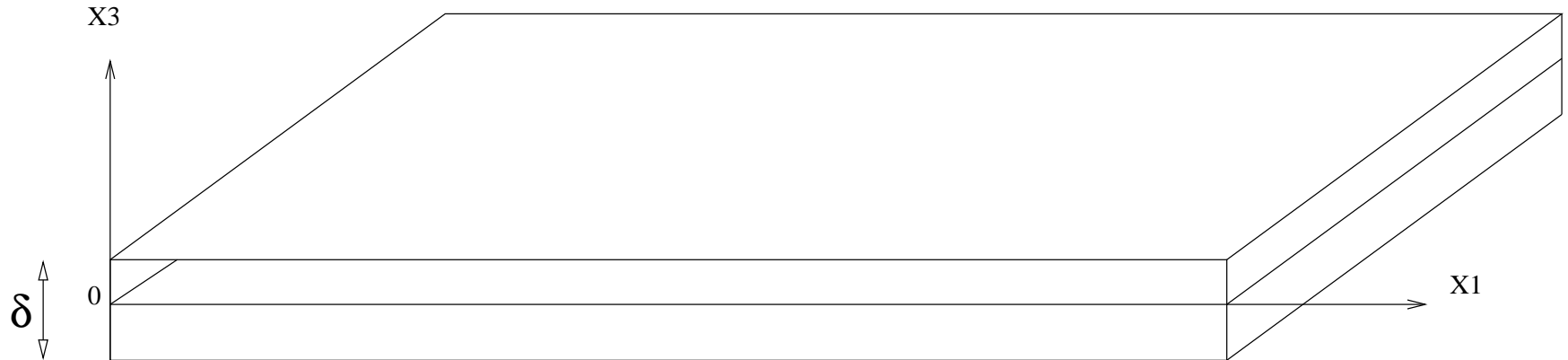


$$Q_\delta = \mathcal{D} \times \left[-\frac{\delta}{2}, \frac{\delta}{2}\right], \quad \delta \rightarrow 0$$

Question :

$$\liminf_{\delta \rightarrow 0} \inf_u \left\{ \int_{Q_\delta} j(e(u)) - \langle F^\delta, u \rangle \right\}?$$

3D-2D asymptotic analysis



$$Q_\delta = \mathcal{D} \times \left[-\frac{\delta}{2}, \frac{\delta}{2}\right], \quad \delta \rightarrow 0$$

Question :

$$\liminf_{\delta \rightarrow 0} \inf_u \left\{ \int_{Q_\delta} j(e(u)) - \langle F^\delta, u \rangle \right\}?$$

Rescaling :

$$y_\alpha = x_\alpha \quad (\alpha = 1, 2), \quad y_3 = \frac{x_3}{\delta}$$

3D-2D asymptotic analysis

$$u_\alpha(x_\alpha, x_3) = v_\alpha(x_a, \frac{x_3}{\delta}), \quad u_3(x_\alpha, x_3) = \frac{1}{\delta} v_3(x_a, \frac{x_3}{\delta}).$$

3D-2D asymptotic analysis

$$u_\alpha(x_\alpha, x_3) = v_\alpha(x_\alpha, \frac{x_3}{\delta}), \quad u_3(x_\alpha, x_3) = \frac{1}{\delta} v_3(x_\alpha, \frac{x_3}{\delta}).$$

$$e(u)(x_\alpha, x_3) = e^\delta(v) := \begin{pmatrix} e_{\alpha,\beta}(v) & \frac{1}{\delta} e_{\alpha,3}(v) \\ \frac{1}{\delta} e_{\alpha,3}(v) & \frac{1}{\delta^2} e_{3,3}(v) \end{pmatrix}$$

3D-2D asymptotic analysis

$$u_\alpha(x_\alpha, x_3) = v_\alpha(x_a, \frac{x_3}{\delta}), \quad u_3(x_\alpha, x_3) = \frac{1}{\delta} v_3(x_a, \frac{x_3}{\delta}).$$

$$e(u)(x_\alpha, x_3) = e^\delta(v) := \begin{pmatrix} e_{\alpha,\beta}(v) & \frac{1}{\delta} e_{\alpha,3}(v) \\ \frac{1}{\delta} e_{\alpha,3}(v) & \frac{1}{\delta^2} e_{3,3}(v) \end{pmatrix}$$

$$F_\alpha^\delta(x_\alpha, x_3) = \frac{1}{\sqrt{\delta}} F_\alpha(x_a, \frac{x_3}{\delta}), \quad F_3^\delta(x_\alpha, x_3) = \sqrt{\delta} F_3(x_a, \frac{x_3}{\delta}),$$

3D-2D asymptotic analysis

$$u_\alpha(x_\alpha, x_3) = v_\alpha(x_a, \frac{x_3}{\delta}), \quad u_3(x_\alpha, x_3) = \frac{1}{\delta} v_3(x_a, \frac{x_3}{\delta}).$$

$$e(u)(x_\alpha, x_3) = e^\delta(v) := \begin{pmatrix} e_{\alpha,\beta}(v) & \frac{1}{\delta} e_{\alpha,3}(v) \\ \frac{1}{\delta} e_{\alpha,3}(v) & \frac{1}{\delta^2} e_{3,3}(v) \end{pmatrix}$$

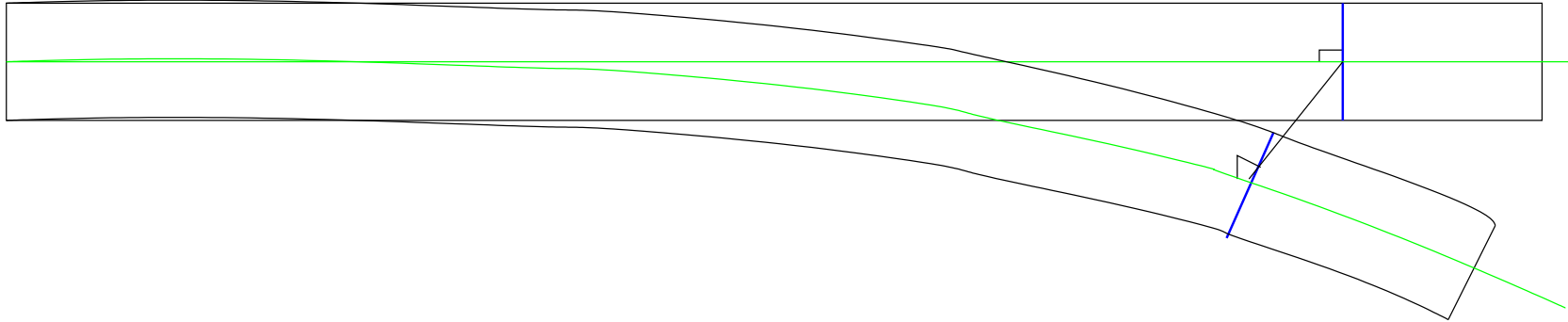
$$F_\alpha^\delta(x_\alpha, x_3) = \frac{1}{\sqrt{\delta}} F_\alpha(x_a, \frac{x_3}{\delta}), \quad F_3^\delta(x_\alpha, x_3) = \sqrt{\delta} F_3(x_a, \frac{x_3}{\delta}),$$

Rescaled problem :

$$\inf_v \left\{ \int_{Q_1} j(e^\delta(v)) - \langle F, v \rangle \right\}$$

3D-2D asymptotic analysis

Definition : $KL := \{v \in H^1(Q_1) / e_{\alpha,3}(v) = 0, e_{3,3}(v) = 0\}$

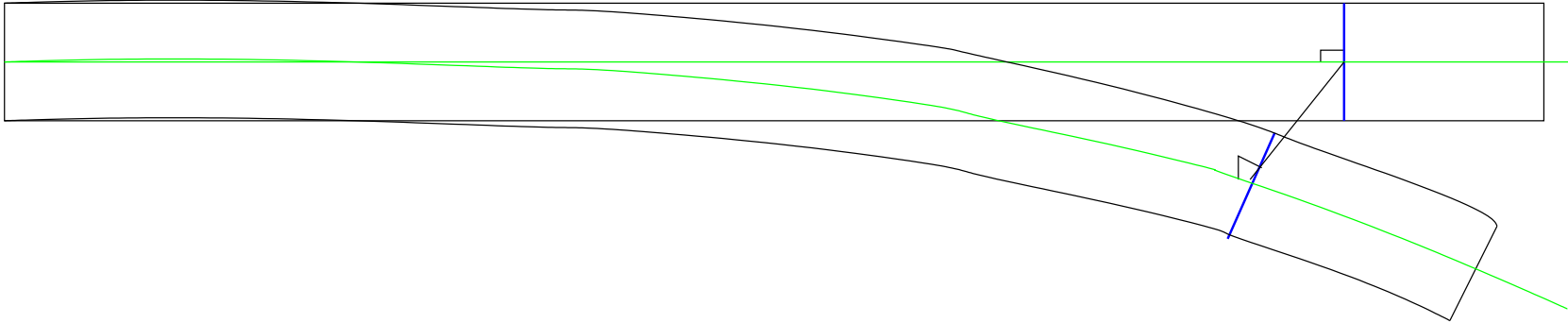


Characterization : v is in KL if and only if there exists w such that $w_\alpha \in H^1(\mathcal{D})$, $w_3 \in H^2(\mathcal{D})$ and

$$v_3(y_\alpha, y_3) = w_3(y_\alpha), \quad v_\alpha(y_\alpha, y_3) = w_\alpha(y_\alpha) - \frac{\partial w_3}{\partial y_\alpha} y_3$$

3D-2D asymptotic analysis

Definition : $KL := \{v \in H^1(Q_1) / e_{\alpha,3}(v) = 0, e_{3,3}(v) = 0\}$



Characterization : v is in KL if and only if there exists w such that $w_\alpha \in H^1(\mathcal{D})$, $w_3 \in H^2(\mathcal{D})$ and

$$v_3(y_\alpha, y_3) = w_3(y_\alpha), \quad v_\alpha(y_\alpha, y_3) = w_\alpha(y_\alpha) - \frac{\partial w_3}{\partial y_\alpha} y_3$$

- replace v by a KL-displacement in the infimum pb.
- explicit the energy in term of w

3D-2D asymptotic analysis

- gives the 2D plate problem :

$$\inf_w \left\{ \int_{\mathcal{D}} \mathcal{W}(e_{\alpha,\beta}(w_\alpha), \nabla^2 w_3) - \bar{F}, w \right\}$$

where

$$\bar{F}_\alpha(y_\alpha) := \int_{-\frac{1}{2}}^{\frac{1}{2}} F_\alpha(y_\alpha, y_3) dy_3$$

$$\bar{F}_3(y_\alpha) := \int_{-\frac{1}{2}}^{\frac{1}{2}} F_3 + y_3 \frac{\partial F_1}{\partial y_1} + x_3 \frac{\partial F_2}{\partial y_2} dy_3$$

and \mathcal{W} is explicit in terms of j .

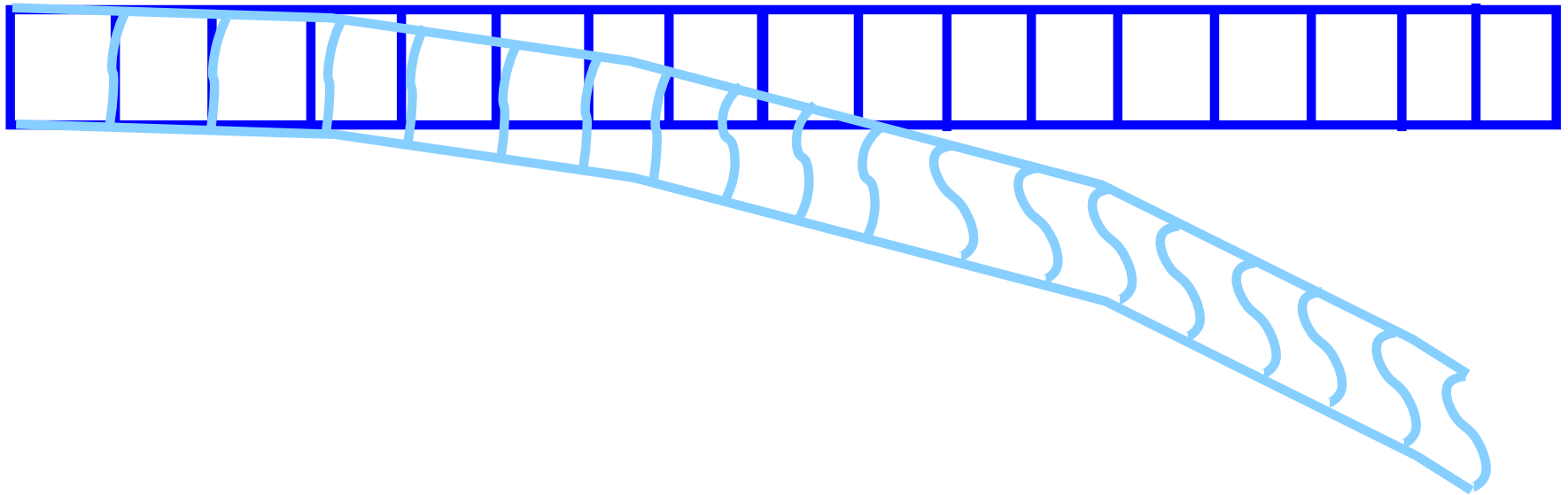
3D-2D asymptotic analysis

- > many extensions (non isotropic, non homogeneous, varying thickness, non linear cases)
- > but no simple extension with high contrast composites (or with voids)

3D-2D asymptotic analysis

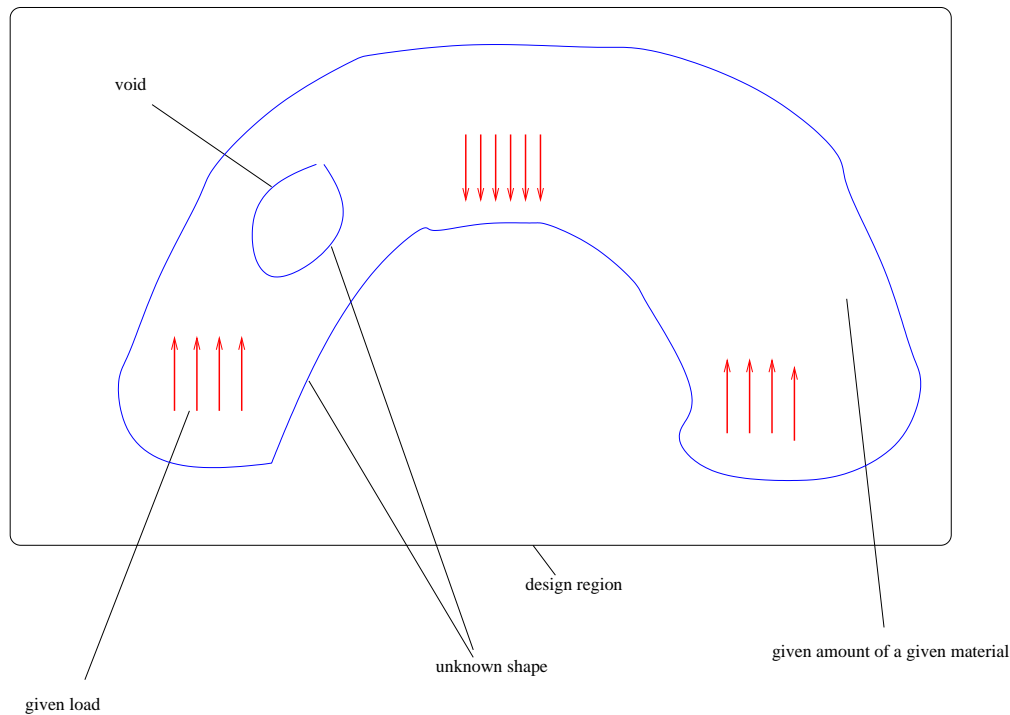
-> many extensions (non isotropic, non homogeneous, varying thickness, non linear cases)

-> but no simple extension with high contrast composites (or with voids)



Shape optimization

In a given design region Q , a given amount $|\Omega|$ of a given material j is placed in order to resist to a given load F (F balanced, $|\text{supp}(F)| = 0$)



Shape optimization

Known : $Q, |\Omega|, j, F \in H^{-1}(Q)$, Unknown : the set (shape) Ω

Goal :

$$\inf_{\Omega} \{C(\Omega, j, F); |\Omega| \text{ fixed}, \Omega \subset Q\}$$

Shape optimization

Known : $Q, |\Omega|, j, F \in H^{-1}(Q)$, Unknown : the set (shape) Ω

Goal :

$$\inf_{\Omega} \{C(\Omega, j, F); |\Omega| \text{ fixed}, \Omega \subset Q\}$$

-> no existence in general.

-> need of a relaxed formulation.

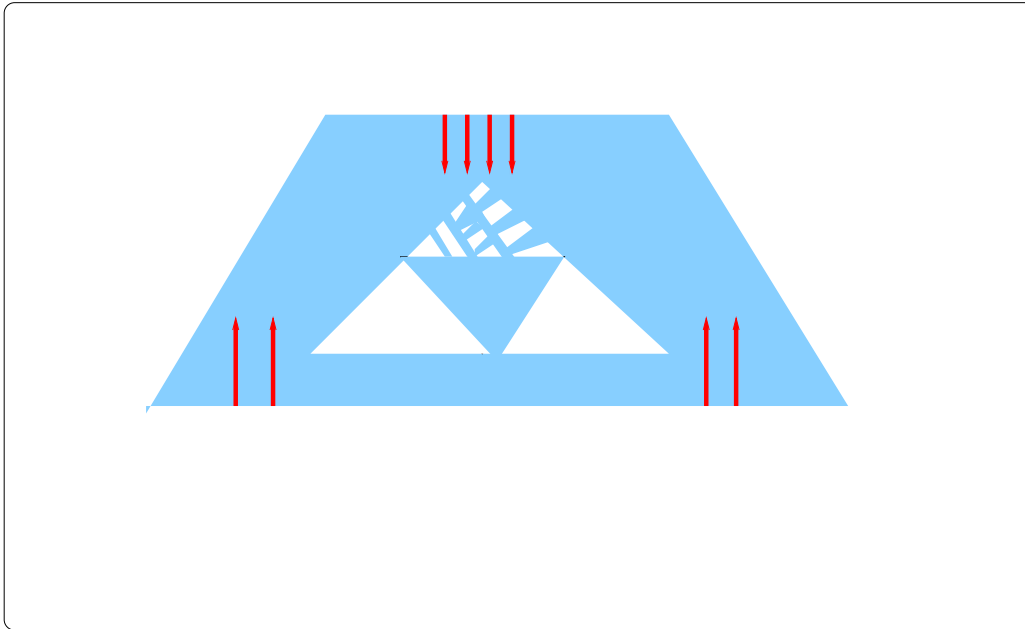
$$C(\Omega, j, F) = \sup_u \left\{ \langle F, u \rangle - \int j(e(u)) 1_{\Omega}(x) dx \right\}$$

$$C(\theta, j, F) = \sup_u \left\{ \langle F, u \rangle - \int j(e(u)) \theta(x) dx \right\}$$

Find an optimal $\theta \in L^{\infty}(Q, [0, 1])$ instead of a characteristic function.

Shape optimization

- > new problem admits a solution $\bar{\theta}$ (fictitious material)
- > $0 < \bar{\theta} < 1$ needs an interpretation...
- > The infimum is not the same as in the original problem !!!



In the homogenized zones (where $0 < \theta < 1$ the resistance is weaker than $\theta j(e(u))$).

Should be replaced by $j^{hom}(\theta, e(u), structure) \dots$ (Allaire)

The considered problem

The optimal shape in an asymptotically thin layer.

$$\inf_{\Omega} \{C(\Omega, j, F^\delta); \Omega \subset Q_\delta, |\Omega| = \tau\delta\}$$

No topological constraint (voids are allowed). It is different from optimizing plates thickness (Bonnetier, Conca)

The considered problem

The optimal shape in an asymptotically thin layer.

$$\inf_{\Omega} \{C(\Omega, j, F^\delta); \Omega \subset Q_\delta, |\Omega| = \tau\delta\}$$

No topological constraint (voids are allowed). It is different from optimizing plates thickness (Bonnetier, Conca)

First let us get rid of the volume constraint (using a Lagrange multiplier).

$$\Phi^\delta(k) = \inf_{\Omega} \{C(\Omega, j, F^\delta) + \frac{k}{\delta}|\Omega|; \Omega \subset Q_\delta\}$$

For a given $k \geq 0$ any solution of the new pb. is a solution of the original one. $|\Omega|$ can be tuned by tuning k . Both problems are equivalent.

Rescaling

- > Same scaling for x, u, F^δ as in plate theory -> y, v, F .
- > Same definition for $e^\delta(v)$.
- > Introduction of $\omega := \{(x_\alpha, \frac{1}{\delta}x_3); x \in \Omega\}$

Rescaling

- > Same scaling for x, u, F^δ as in plate theory -> y, v, F .
- > Same definition for $e^\delta(v)$.
- > Introduction of $\omega := \{(x_\alpha, \frac{1}{\delta}x_3); x \in \Omega\}$

$$\Phi^\delta(k) = \inf_{\omega} \left\{ C^\delta(\omega) + \frac{k}{\delta} |\omega|; \Omega \subset Q_1 \right\}$$

$$C^\delta(\omega) := \sup_u \left\{ \langle F, u \rangle - \int_{Q_1} j(e^\delta(u)) 1_\omega(x) dx \right\}$$

Rescaling

- > Same scaling for x, u, F^δ as in plate theory -> y, v, F .
- > Same definition for $e^\delta(v)$.
- > Introduction of $\omega := \{(x_\alpha, \frac{1}{\delta}x_3); x \in \Omega\}$

$$\Phi^\delta(k) = \inf_{\omega} \left\{ C^\delta(\omega) + \frac{k}{\delta} |\omega|; \Omega \subset Q_1 \right\}$$

$$C^\delta(\omega) := \sup_u \left\{ \langle F, u \rangle - \int_{Q_1} j(e^\delta(u)) 1_\omega(x) dx \right\}$$

or in a dual way

$$C^\delta(\omega) = \inf_{\sigma} \left\{ \int_{Q_1} j^*(\Pi^\delta(\sigma)); \sigma = 0 \text{ on } Q_1 \setminus \omega, \operatorname{div} \sigma + F = 0 \right\}$$

$$\Pi^\delta(\sigma) := \begin{pmatrix} \sigma_{\alpha,\beta} & \delta\sigma_{\alpha,3} \\ \delta\sigma_{\alpha,3} & \delta^2\sigma_{3,3} \end{pmatrix}$$

Remark

Indeed

$$\int j^*(\Pi^\delta(\sigma)) + j(e^\delta(u)) \geq \int \Pi^\delta(\sigma) \cdot e^\delta(u) = \int \sigma \cdot e(u) = - \langle F, u \rangle$$

Description of the limit problem

Let \bar{j} be the energy in term of displacement for plane stress.

$$\bar{j}(A) := \inf_a j \left(\begin{array}{ccc} & A & a_1 \\ & & a_2 \\ a_1 & a_2 & a_3 \end{array} \right)$$

Description of the limit problem

Let \bar{j} be the energy in term of displacement for plane stress.

$$\bar{j}(A) := \inf_a j \left(\begin{pmatrix} & & a_1 \\ & A & a_2 \\ a_1 & a_2 & a_3 \end{pmatrix} \right)$$

The limit energy is described in term of the density θ of material in Q_1 (as in the fictitious material case).

$$C(\theta) = \sup_{u \in KL} \left\{ \langle F, u \rangle - \int_{Q_1} \bar{j}(e_{\alpha,\beta}(u)) \theta(x) dx \right\}$$

Description of the limit problem

or in a dual way

$$C(\theta) := \inf_{\sigma} \left\{ \int_{Q_1} \theta^{-1} \times (\bar{j})^*(\sigma), \operatorname{div}_{\alpha} [[\sigma]] + \bar{F}_{\alpha} = 0, \operatorname{div}_{\alpha} \operatorname{div}_{\alpha} [[y_3 \sigma]] + \bar{F}_3 = 0 \right\}$$

where σ is a 2×2 matrix and $[[\cdot]]$ means the mean value on a section.

$$[[\phi]](y_{\alpha}) = \int_{-\frac{1}{2}}^{\frac{1}{2}} \phi(y_{\alpha}, y_3) dy_3$$

Description of the limit problem

or in a dual way

$$C(\theta) := \inf_{\sigma} \left\{ \int_{Q_1} \theta^{-1} \times (\bar{j})^*(\sigma), \operatorname{div}_{\alpha} [[\sigma]] + \bar{F}_{\alpha} = 0, \operatorname{div}_{\alpha} \operatorname{div}_{\alpha} [[y_3 \sigma]] + \bar{F}_3 = 0 \right\}$$

where σ is a 2×2 matrix and $[[\cdot]]$ means the mean value on a section.

$$[[\phi]](y_{\alpha}) = \int_{-\frac{1}{2}}^{\frac{1}{2}} \phi(y_{\alpha}, y_3) dy_3$$

Indeed

$$\begin{aligned} & \int \theta^{-1} \times (\bar{j})^*(\sigma) + (\bar{j})(e_{\alpha, \beta}(u) \theta) - \langle F, u \rangle \\ &= \int \theta^{-1} \times ((\bar{j})^*(\sigma) + (\bar{j})(\theta e_{\alpha, \beta}(u))) - \langle F, u \rangle \\ &\geq \int \sigma \cdot e_{\alpha, \beta}(u) - \langle F, u \rangle = \int -\operatorname{div}_{\alpha}(\sigma) \cdot u - \langle F, u \rangle \end{aligned}$$

which vanishes if $\operatorname{div}_{\alpha}(\sigma) + F$ is orthogonal to $KL \dots$

Description of the limit problem

The limit problem reads

$$\Phi(k) = \inf_{\theta} \{ C(\theta) + k \int_{Q_1} \theta \}$$

$$\Phi(k) = \inf_{\theta} \sup_{u \in KL} \{ \langle F, u \rangle - \int_{Q_1} \bar{j}(e(u))\theta + k \int_{Q_1} \theta \}$$

Description of the limit problem

The limit problem reads

$$\Phi(k) = \inf_{\theta} \left\{ C(\theta) + k \int_{Q_1} \theta \right\}$$

$$\Phi(k) = \inf_{\theta} \sup_{u \in KL} \left\{ \langle F, u \rangle - \int_{Q_1} \bar{j}(e(u))\theta + k \int_{Q_1} \theta \right\}$$

Theorem :

(i) the limit problem admits a solution.

(ii) $\lim_{\delta \rightarrow 0} \Phi^{\delta}(k) = \Phi(k)$

(iii) any minimizing sequence ω^{δ} converges weakly to a solution $\bar{\theta}$ of the limit pb.*

(iv) the limit problem admits the following equivalent formulations.

Equivalent formulations

$$\Phi(k) = \inf_{\theta} \sup_{u \in KL} \left\{ \langle F, u \rangle - \int_{Q_1} \bar{j}(e(u))\theta + k \int_{Q_1} \theta \right\}$$

Equivalent formulations

$$\Phi(k) = \inf_{\theta} \sup_{u \in KL} \left\{ \langle F, u \rangle - \int_{Q_1} \bar{j}(e(u))\theta + k \int_{Q_1} \theta \right\}$$

Inverting inf and sup and computing explicitly the infimum in θ leads to the 3D formulation in terms of displacement only :

$$\Phi(k) = \sup_{u \in KL} \left\{ \langle F, u \rangle - \int_{Q_1} (\bar{j}(e(u)) - k)_+ \right\}$$

Equivalent formulations

$$\Phi(k) = \inf_{\theta} \sup_{u \in KL} \left\{ \langle F, u \rangle - \int_{Q_1} \bar{j}(e(u))\theta + k \int_{Q_1} \theta \right\}$$

Inverting inf and sup and computing explicitly the infimum in θ leads to the 3D formulation in terms of displacement only :

$$\Phi(k) = \sup_{u \in KL} \left\{ \langle F, u \rangle - \int_{Q_1} (\bar{j}(e(u)) - k)_+ \right\}$$

or 3D formulation in terms of stress only

$$\Phi(k) = \inf_{\sigma} \left\{ \int_{Q_1} ((\bar{j} - k)_+)^*(\sigma); \operatorname{div}_{\alpha}[[\sigma]] + \bar{F}_{\alpha} = 0, \operatorname{div}_{\alpha} \operatorname{div}_{\alpha}[[x_3 \sigma]] + \bar{F}_3 = 0 \right\}$$

Equivalent formulations

As u is in KL , it can be explicitated in terms of a 2D displacement w :

$$\Phi(k) = \sup_w \{ \langle \bar{F}, w \rangle - \int_{\mathcal{D}} W_k(e_{\alpha,\beta}(w_\alpha), \nabla_{\alpha,\beta}^2(w_3)) \}$$

$$W_k(A, B) := \int_{-\frac{1}{2}}^{\frac{1}{2}} (\bar{j}(A - x_3 B) - k)_+$$

Equivalent formulations

As u is in KL , it can be explicited in terms of a 2D displacement w :

$$\Phi(k) = \sup_w \{ \langle \bar{F}, w \rangle - \int_{\mathcal{D}} W_k(e_{\alpha,\beta}(w_\alpha), \nabla_{\alpha,\beta}^2(w_3)) \}$$

$$W_k(A, B) := \int_{-\frac{1}{2}}^{\frac{1}{2}} (\bar{j}(A - x_3 B) - k)_+$$

The dual form of the last pb. involves 2D-generalized stresses.

$$\Phi(k) = \inf_{\lambda, \eta} \left\{ \int_{\mathcal{D}} W_k^*(\lambda, \eta); \operatorname{div} \lambda + \bar{F}_\alpha = 0; \operatorname{div} \operatorname{div} \eta + \bar{F}_3 = 0 \right\}$$

Equivalent formulations

As u is in KL , it can be explicited in terms of a 2D displacement w :

$$\Phi(k) = \sup_w \{ \langle \bar{F}, w \rangle - \int_{\mathcal{D}} W_k(e_{\alpha,\beta}(w_\alpha), \nabla_{\alpha,\beta}^2(w_3)) \}$$

$$W_k(A, B) := \int_{-\frac{1}{2}}^{\frac{1}{2}} (\bar{j}(A - x_3 B) - k)_+$$

The dual form of the last pb. involves 2D-generalized stresses.

$$\Phi(k) = \inf_{\lambda, \eta} \{ \int_{\mathcal{D}} W_k^*(\lambda, \eta); \operatorname{div} \lambda + \bar{F}_\alpha = 0; \operatorname{div} \operatorname{div} \eta + \bar{F}_3 = 0 \}$$

(v) all these formulations have solutions $(\bar{u}, \bar{\sigma}, \bar{w}, \bar{\lambda}, \bar{\eta})$.

(vi) The optimal solutions correspond.

(vii) for any minimizing sequence, $\Pi^\delta(\sigma^\delta(\omega^\delta))$ converges weakly* to $\Pi^0(\bar{\sigma})$.

Optimality conditions

• $\bar{u} \in KL.$

Optimality conditions

- $\bar{u} \in KL.$
- $div_\alpha[[\bar{\sigma}]] + \bar{F}_\alpha = 0, \quad div_\alpha div_\alpha[[x_3\bar{\sigma}]] + \bar{F}_3 = 0$

Optimality conditions

- $\bar{u} \in KL.$
- $div_\alpha [[\bar{\sigma}]] + \bar{F}_\alpha = 0, \quad div_\alpha div_\alpha [[x_3 \bar{\sigma}]] + \bar{F}_3 = 0$
- $\bar{\sigma} = \bar{\theta} \frac{\partial \bar{j}}{\partial e_{\alpha,\beta}}(e_{\alpha,\beta}(\bar{u}))$

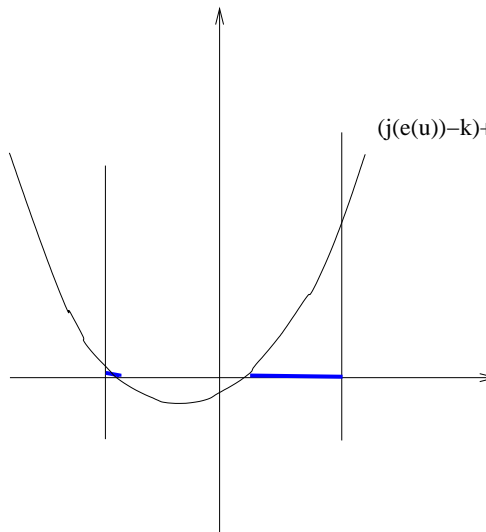
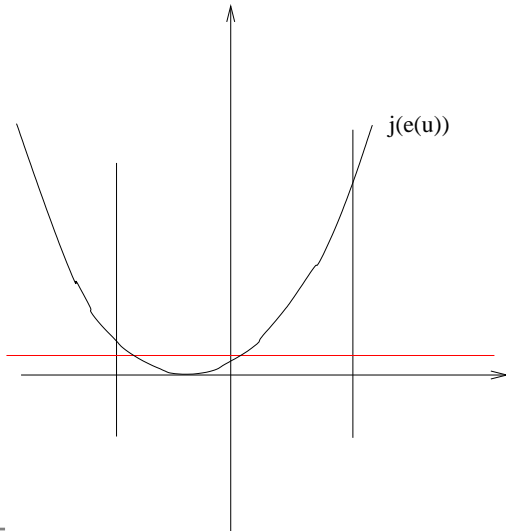
Optimality conditions

- $\bar{u} \in KL.$
- $div_\alpha[[\bar{\sigma}]] + \bar{F}_\alpha = 0, \quad div_\alpha div_\alpha[[x_3\bar{\sigma}]] + \bar{F}_3 = 0$
- $\bar{\sigma} = \bar{\theta} \frac{\partial \bar{j}}{\partial e_{\alpha,\beta}}(e_{\alpha,\beta}(\bar{u}))$
- $\bar{\theta} \times (\bar{j}(e_{\alpha,\beta}(\bar{u})) - k) = (\bar{j}(e_{\alpha,\beta}(\bar{u})) - k)_+$

Optimality conditions

- $\bar{u} \in KL.$
- $div_\alpha [[\bar{\sigma}]] + \bar{F}_\alpha = 0, \quad div_\alpha div_\alpha [[x_3 \bar{\sigma}]] + \bar{F}_3 = 0$
- $\bar{\sigma} = \bar{\theta} \frac{\partial \bar{j}}{\partial e_{\alpha,\beta}} (e_{\alpha,\beta}(\bar{u}))$
- $\bar{\theta} \times (\bar{j}(e_{\alpha,\beta}(\bar{u})) - k) = (\bar{j}(e_{\alpha,\beta}(\bar{u})) - k)_+$

But $e_{\alpha,\beta}(\bar{u}) = e_{\alpha,\beta}(\bar{w}) + \frac{\partial^2 \bar{w}_3}{\partial x_\alpha \partial x_\beta} \times x_3$ is linear

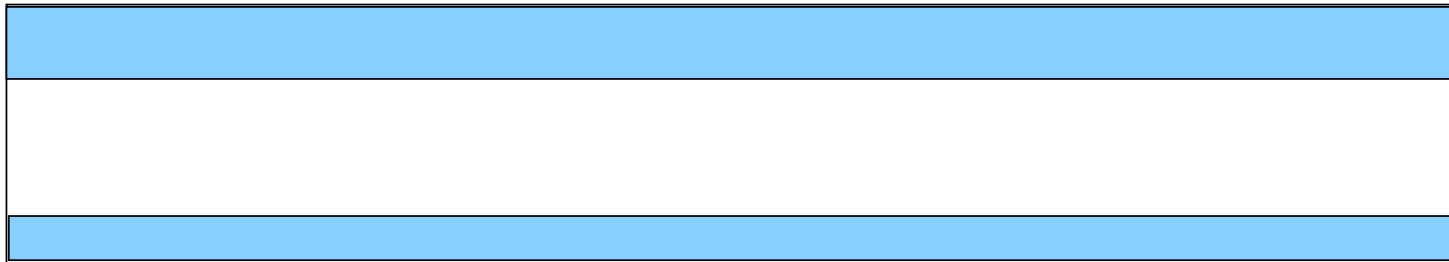


Optimal structures

Assume that $\frac{\partial^2 \bar{w}_3}{\partial x_\alpha \partial x_\beta} \neq 0$ (generic case).

Then, at point y_α , $\theta(y_\alpha, y_3)$ is the **characteristic function** of the complementary of an interval..

The optimal structure is a **real shape** made of **two plates** with varying thickness **along the top and bottom** of the design domain.



Optimal structures

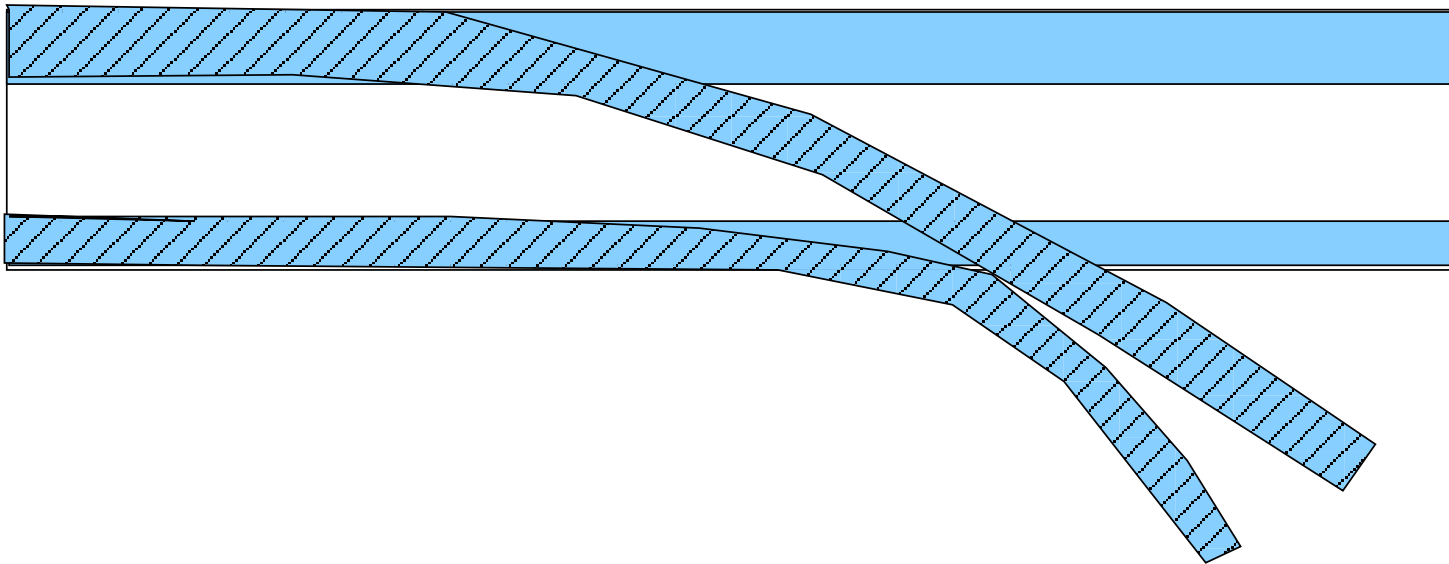
From the mechanical point of view :

- concentrating the material on the top and the bottom is usual (T-shape or I-shape for beam, sandwiches-shape for plates)
- but using two disconnected plates seems idiot.

Optimal structures

From the mechanical point of view :

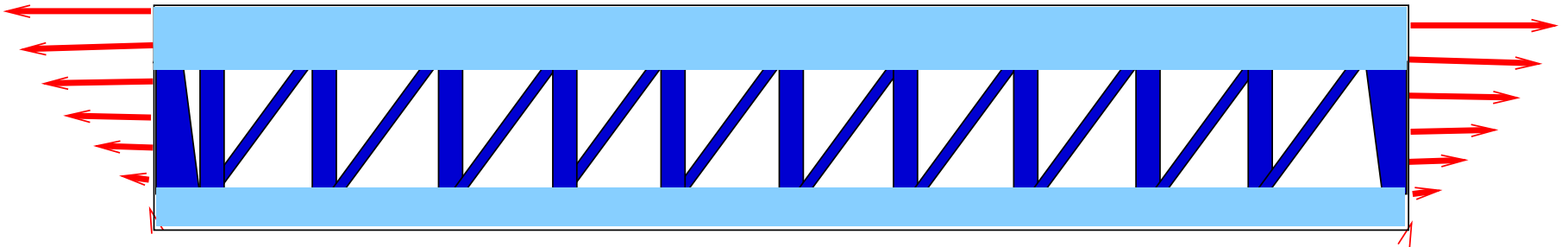
- concentrating the material on the top and the bottom is usual (T-shape or I-shape for beam, sandwiches-shape for plates)
- but using two disconnected plates seems idiot.



Optimal structures

The point is that a negligible amount of material is needed for connecting the two plates (it disappears at the limit).

- The same is true for the boundary layers needed for taking into account the applied forces.

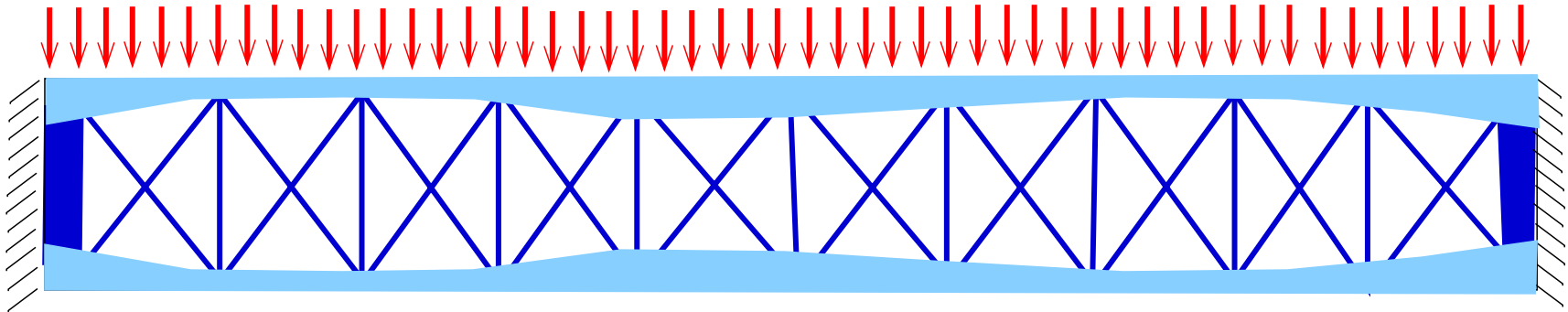


Note that the relative thickness of the two layers depends on $e_{\alpha,\beta}(\bar{w})$.

Optimal structures

The thickness depends on x_α through $\frac{\partial^2 \bar{w}_3}{\partial x_\alpha \partial x_\beta}$ which varies from place to place.

Example of the “bridge” :



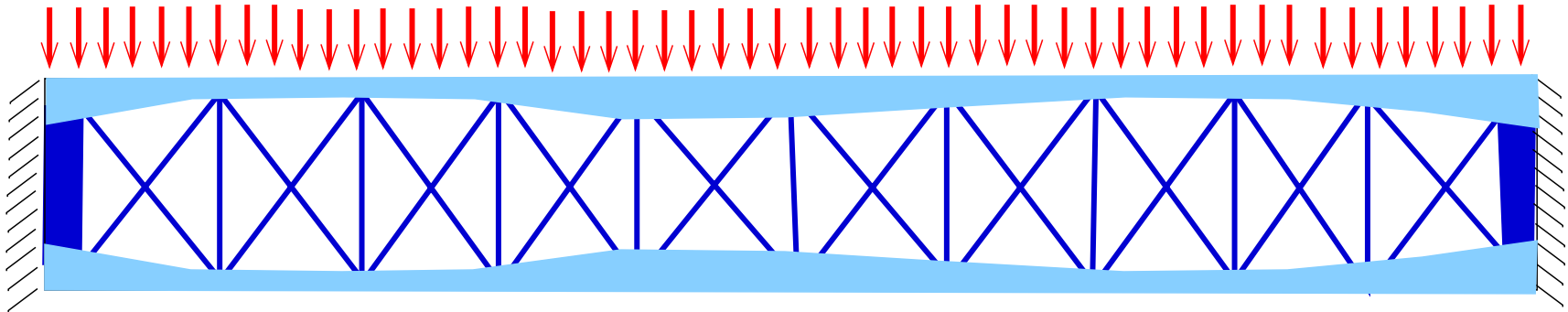
Note that the dark region is not given by the limit problem. It's a “decoration”.

The fact that it is a higher order problem likely explains why there are so many different shapes used for this intermediate part.

Optimal structures

The thickness depends on x_α through $\frac{\partial^2 \bar{w}_3}{\partial x_\alpha \partial x_\beta}$ which varies from place to place.

Example of the “bridge” :



Note that the dark region is not given by the limit problem. It's a “decoration”.

The fact that it is a higher order problem likely explains why there are so many different shapes used for this intermediate part.